



GICL、GIICL系列 鼓形齿式联轴器

Curved Tooth Coupling GICL & GIICL

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1. 概述

1.1 鼓形齿式联轴器是由齿数相同的沿齿长方向的齿厚为鼓形的外齿轴套和带凸缘的内齿圈等零件组成的一种挠性联轴器，用于联接有径向位移和轴线偏角的同轴线两轴同步回转以传递运动和转矩。

1.2 我公司生产的鼓形齿式联轴器具有如下特点：

1.2.1 外齿轮套的齿厚呈鼓形，中间厚两端薄，可避免齿端棱角接触，因此允许两轴线有较大的角位移，一般设计为 $\pm 1.5^\circ$ ，特殊设计为 3° ；

1.2.2 由于鼓形齿有效地避免了轮齿端部接触引起的应力集中，因此能承受较大的转矩和冲击载荷，过载能力大，工作平稳可靠；在相同的角位移时，比直齿联轴器的承载能力提高 $15 - 20\%$ ；

1.2.3 鼓形齿的轮齿啮合参数和整体结构经过优化设计，外形尺寸小，重量轻，转动惯量小，传动效率高达 99.7% ；

1.2.4 鼓形齿式联轴器可正、反方向回转；

1.2.5 鼓形齿式联轴器不承受轴向力，结构上能补偿运转中的少量轴向位移；

1.2.6 由于外齿轮套的轮齿外圆制成球面因此易于安装和调整；

1.2.7 鼓形齿式联轴器既可水平安装，也可设计成立式安装的形式；但不适宜用于要求减振、缓冲以及两轴严格对中的场合。若需立式安装的，请在订货时说明；

1.2.8 由于选用了优质合金钢锻件，合理的齿部硬化工艺，以及良好的润滑设计，因此我公司生产的鼓形齿式联轴器具有承载能力大、耐磨损、噪声低、工作寿命长、使用维护简单等优点。

1. General

1.1 Curved Tooth coupling consists of the hub with external Curved along teeth length direction teeth and the flanged internal gearing with the same number of teeth. It is used for coupling and transmission of synchronized rotation and torque between two coaxial shafts with a certain radial and angular misalignment

1.2 The Curved Tooth coupling produced by us has the following features:

1.2.1 The hub with external teeth has Curved teeth which allow angular displacement between two axial lines as high as $\pm 1.5^\circ$ for ordinary design and as 3° and above for special design.

1.2.2 It is able to withstand relatively high torque and impact load and has rather high overload capacity since the Curved Tooth can prevent concentrated stress and thus operation is smooth and reliable. The load capacity is higher than the gear coupling with straight teeth by 15-20% under the same angular misalignment.

1.2.3 Design of the meshing parameters of the Curved Tooth and the entire construction is optimized and thus the coupling has compact overall dimensions, light weight and lower moment of inertia. Transmission efficiency is as high as 99.7%.

1.2.4 The Curved Tooth coupling may rotate in two directions and any one side may be used as driven input end.

1.2.5 The Curved Tooth coupling can't withstand axial load and only a small axial displacement can be compensated by the construction during running.

1.2.6 It can be easily assembled and adjusted due to the curved (spherical) teeth.

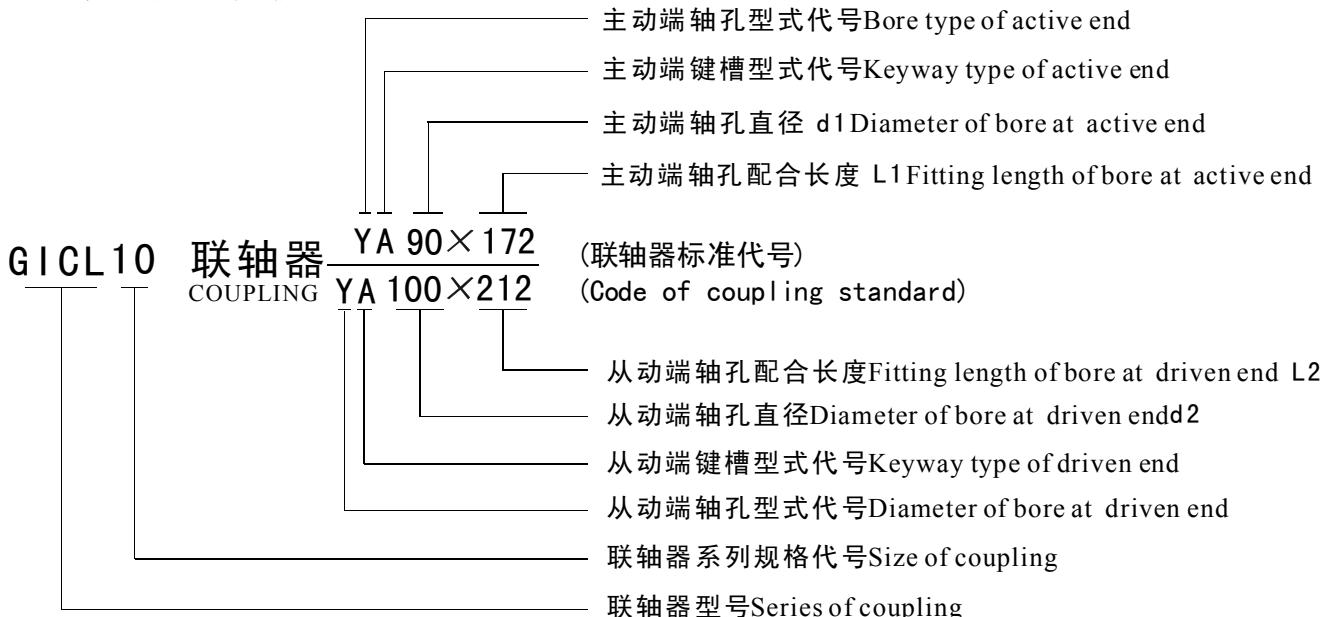
1.2.7 Curved Tooth coupling may be designed for either horizontal or vertical installation. However, it is not suitable for applications requiring cushioning and buffering and strict alignment of the two axes.

1.2.8 The Curved Tooth coupling by our company has the advantages such as high load capacity, high wear-resistance, low noise level, long service life, and simple maintenance resulted from using high quality alloy steel forging stocks, rational teeth hardening process and perfect lubrication design.

2. 订货须知

2. Ordering code

2.1、订购鼓形齿式联轴器时，
规定的标记方法如下：



2.2 标记示例：欲订购**GICL10**型鼓形齿式联轴器，
已知：
主动端：Z1型轴孔，C型键槽，d1=90mm，
L1=132mm，
从动端：Y型轴孔，A型键槽，d2=95 mm， L2 =
177 mm，

标记为：

GICL10联轴器
Coupling GICL10

Z1C 90×132
YA 95×177

当主、从动端轴孔的配合长度**L1**、**L2**都符合选型表中**L**的标准值时，可以省略标注配合长度。但我们建议不管主、从动端轴孔的配合长度**L1**、**L2**是否符合选型表中**L**的标准尺寸值，都应完整地标注联轴器的型号、规格和尺寸。

2.2 Ordefinge example:: if a Curved tooth coupling of type GICL10 is requested and the following data are known:
Driven end: Shaft Bore Z1 type, Keyway is C type, d1=90mm, L1=132mm
Idle end: Bore type Y, Keyway A type, d2=95mm and L2=177mm
Then the ordering code is:

the bore fitting length of both driven and idle ends L1 and L2 can be omitted in the ordering code if both of them from the table are standard. However we recommend that the fitting length of bores L1 add L2 are always indicated in the Ordering code.

表1：圆柱形轴孔与轴伸的配合

Table 1: Fitting of the cylindrical bore and shaft extension

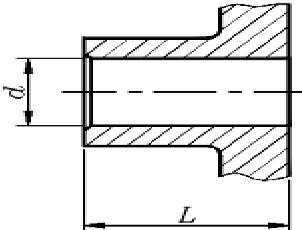
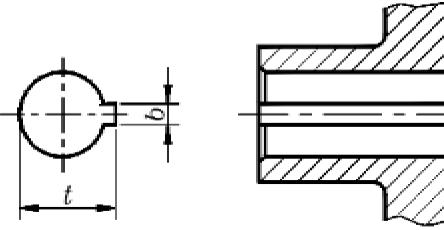
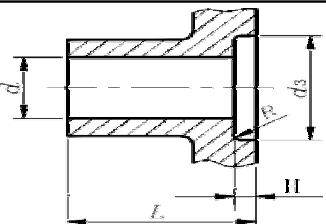
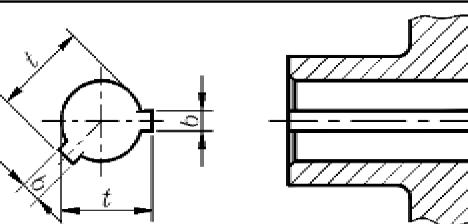
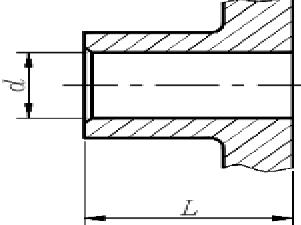
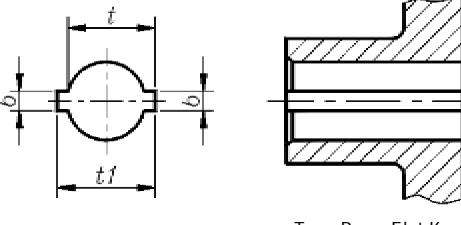
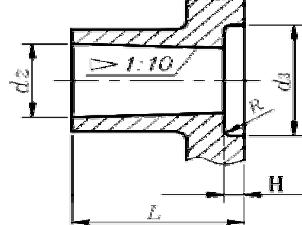
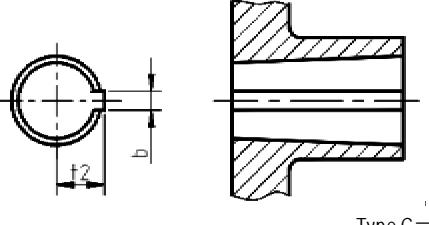
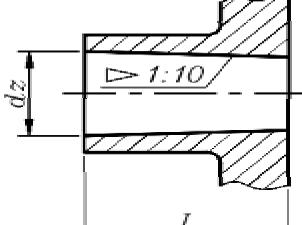
直径Diameter d(mm)	配合代号 Fitting code	
> 6—30	H7/j6	根据使用要求，也可选用如下配合 Can also be used depending on application requirement
>30—50	H7/k6	
>50	H7/m6	H7/n6 H7/p6 H7/r6

注：选用过盈大于表中的配合时，应验算半联轴器轮毂的强度。采用无键过盈联接配合，按要求另行确定。

Note: If the interference of tight fit is bigger than listed in above table 1, Intension of the hubs of coupling have to be checked by calculate. if fitting does not by key, the fitting code should be consulted.

表2 联轴器轴孔键槽及其联接的型式和代号

Table 2 Type and Code of Coupling Bore, keyway and its Connection

轴孔型式和代号 Shaft Bore Type & Code	键槽型式和代号 Connection Type and Code
 Y型-长圆柱形轴孔 Type Y—Long Cylindrical Bore	 A型-平键单键槽 Type A—Flat Key, Single Keyway
 J型-有沉孔的短圆柱形轴孔 Type J—Short Cylindrical Bore with Counter Bore	 B型-120° 布置平键双键槽 Type B—Flat Key, Double Keyways Apart from 120° Each Other
 J ₁ 型-无沉孔的短圆柱形轴孔 Type J ₁ —Short Cylindrical Bore without Counter Bore	 B ₁ 型-180° 布置平键双键槽 Type B ₁ —Flat Key, Double Keyway Apart from 180° Each Other
 Z型-有沉孔的长圆锥形轴孔 Type Z—Long Tapered Bore with Counter Bore	 C型-圆锥形轴孔平键单键槽 Type C—Flat Key, Single Key way for Tapered Bore
 Z ₁ 型-无沉孔的长圆锥形轴孔 Type Z ₁ —Long Tapered Bore without Counter Bore	矩形花键联接轴孔按GB/T 1444 圆柱直齿渐开线花键按GB/T3478.1 Square Splined Bore in accordance with GB/T 1444. Cylindrical Evolute Spline in accordance with GB/T 3478.1

约定：圆柱形轴孔的尺寸精度为H7，键槽宽度b选用GB1095-1979《平键 键槽的剖面尺寸》标准值，宽度b的尺寸精度为该标准规定的Js9。圆锥形轴孔的键槽型式和尺寸则按GB/T3852-1997《联轴器轴孔和联接型式及尺寸》的规定。若用户所需轴孔、键槽的尺寸和配合与上述约定不符，务必在订货时说明。

Note: If there is no special indication fitting accuracy of cylindrical shaft bore is H7, the width "b" of the keyway is selected as standard value from GB1095<Section Dimensions of the key, and key way>, and the tolerance of width "b" is specified as Js9. The keyway type and dimensions of tapered bore is specified in GB/T3852-1997 <Type and Dimensions of shaft bore and connection of the coupling>. If the requested dimensions and fitting of shaft bore and keyway do not meet the above mentioned standards it must be noted when ordering.

3、选型指南

3.1 选用程序

正确选择能满足使用目的、并能安全运转的联轴器，需要考虑的因素很多，主要有以下几点：

3.1.1 选用标准联轴器 我公司生产的联轴器都是成熟的、符合国家和机械行业标准的联轴器，而选型样本中尚无国家或行业标准代号的联轴器，则是我公司参考引进样机和国外标准消化吸收和自行研制的，都已经过实际使用考验。这些标准联轴器的结构型式和规格基本上能满足不同转矩、转速和工况条件的要求。

3.1.2 选择联轴器品种、型式 根据联轴器在传动系统中的综合功能以及配套主机的需要选择联轴器的结构型式。例如：当长距离传动、联接的轴向尺寸较大时，宜选用接中间轴型或接中间套型联轴器；当联轴器与制动器配套使用时，则宜选用带制动轮或制动盘型式的联轴器；等等。

3.1.3 联轴器的转矩计算 在计算联轴器的传递转矩时，必须考虑传动装置的负荷状态、转速以及两轴轴线偏移量等因素。原则上，传动系统中动力机的功率应大于工作机所需功率，低速端的转矩应大于高速端的理论转矩；具体计算见下所述。

3.1.4 初选联轴器型号 根据计算转矩，从标准系列中可选定相近似的公称转矩，初步选定联轴器的型号。所谓公称转矩，是指根据系列化要求，设计每一规格联轴器所能长期传递的转矩。

3.1.5 根据轴径调整型号 初步选定的联轴器联接尺寸，应符合主、从动端轴径的要求，否则应根据轴径d调整联轴器的规格。当主、从动端轴径不相同时，应按大轴径选择联轴器的型号。

3. Guide for type selection

3.1 Selection procedure

Correct selection of the coupling allows it to meet the application requirement and to be safety run. For this reason many factors have to be considered among which the major items are as follows:

3.1.1 Selection of standard coupling

The couplings produced by our company as successful products which are in compliance with National Mechanical Trade Standards. However those that are not yet the national or trade standard in the catalogue are coupling that our company introduced from abroad and designed and manufactured which have been proved in practical service. These coupling can meet requirement for different torque, revolution and duty cycle with their different constructions and sizes.

3.1.2 Selection of type

Construction type of coupling is to be selected in accordance with the comprehensive function of the coupling in the transmission system and the requirement of the machine for which the coupling serves. For example, when transmission is provided for a long distance and the axial dimension of connection is rather large the coupling with intermediate shaft or intermediate sleeve should be selected, when the coupling is to be used in conjunction with brake the coupling with brake wheel or brake disc should be selected, etc.

3.1.3 Torque calculation of the coupling

In calculation of the transmitted torque of the coupling the factors such as loading condition and speed of the transmission unit, the misalignment between the two axial lines have to be taken into account. In principle, the horsepower of the prime mover should be higher than that required by operation of the machine, the torque at lower speed end should be higher than theoretical torque at high speed end. Calculation is shown below.

3.1.4 Primary selection of the coupling model

According to calculated torque a closer nominal torque can be chosen from the standard series and primarily select a coupling model. The nominal torque is referred to a torque for which the coupling is designed to transmit for a long term.

3.1.5 Adjustment of the Model According to Shaft Diameter

The primarily selected coupling should have dimensions coupling with shaft diameters at both the driven and idle ends. otherwise the coupling size should be adjusted in accordance with shaft diameter. Where the shaft diameters at driven and idle ends, are different the coupling size shall be selected in accordance with the larger one.

3.1.6 选择联接型式

联轴器联接型式的选择，取决于主、从动端与轴的联接型式，一般多采用键联接，用得较多的是A型键联接。

3.1.7 选定联轴器品种、型式、规格（型号）

为了保证轴和键的强度，在选定了联轴器型号（规格）后，应对轴和键的强度作校核验算，以最后确定联轴器的型号。

3.2 联轴器传递转矩的计算

联轴器的传递转矩应取机械不稳定运转时的动载荷及过载状态下的最大转矩，同时要考虑两轴轴线的偏移量及工作转速等因素。如果不能精确计算时，传递转矩可按以下方法求得。

3.2.1 联轴器的理论转矩

根据动力机驱动功率和工作转速计算而得，即：

$$T = 9550 \frac{P_w}{n}$$

式中：

T——理论转矩，Nm；

Pw——驱动功率，kW；

n——工作转速，r/min。

3.2.2 联轴器的计算转矩 由理论转矩和动力机系数、工况系数及其它有关系数计算而得，即：

$$T_c = T K_w K_z$$

式中：

Tc——计算转矩，Nm；

Kw——动力机系数，根据动力机类别不同，

将动力机分成四类，见附录表1；

K——工况系数，考虑传动系统由于载荷变化、冲击载荷、工作环境等因素对联轴器在实际传递转矩时的影响系数，标准工作状态下的平均值见附录表5；

3.1.6 Selection of connection type

Connection type of coupling is selected depending on connection type with the shafts at driven and idle ends.

Usually key type connection is used and more commonly "A" type of key connection is used.

3.1.7 Selection of the coupling type and model

After the coupling type, model (size) have been selected the strength of the shaft and the key has to be checked to finally determine the coupling size.

3.2 Calculation of coupling transmitted torque

Coupling transmitted torque shall be taken as the max torque under unstable running of the machine and under dynamic load and overload condition, and at the same time the displacement between the two axial lines and running speed etc. have to be considered. If it is impossible to calculate accurately, the transmitted torque can be obtained thorough the following method.

3.2.1 Theoretical torque of coupling

Based on the driving power of prime mover and operation speed the torque will be

Where:
T---Theoretical torque, in Nm
Pw--- Driving power, in kW
n --- Running speed, in r/min

3.2.2 Calculation torque of coupling

The torque is obtained based on theoretical torque, drive prime mover factor, duty factor and other factors:

Where:
Tc -- Calculation torque, in Nm

Kw -- Prime mover factor, it depends on the prime mover type (see A.1)

K -- Duty factor, considering the effect of transmission system on the actual transmitted torque due to load variation, impact load, environmental circumstances etc. its average values under standard running condition are shown in A.5,

K_Z——起动系数，主动端起动频率Z形成的附加负载，见附录表2；

3.3 联轴器转矩与转速的修正计算

联轴器标准系列表中的公称转矩和许用转速是在被联接两轴轴间偏角 $\Delta \alpha = 0^\circ$ 和恒载荷的情况下确定的。当联轴器的实际转速超过规定的许用转速，以及被联接两轴可能产生两轴轴线偏移时，应取较小的许用传递转矩。鼓形齿式联轴器的计算转矩按下列式修正：

式中：

T_n——联轴器的公称转矩，Nm；

K_t——转矩修正系数，按图1中的线图查取；

K_Z -- Starting factor, additional load imposed due to starting frequency Z, see A.2.

3.3 Corrected calculation of coupling torque and speed

In the coupling standard series table the nominal torque and permissible revolution(speed) is determined at angular misalignment $\Delta \alpha$ between two coupled shaft $\Delta \alpha = 0^\circ$ and under constant load. Where the actual speed of sible transmitted torque shall be taken. The calculated torque of flexible gear coupling is to be corrected as follows:

$$T_c \leq K_t T_n$$

Where:

T_n-- Nominal torque of coupling, in Nm

K_t-- Correction factor of torque, and is taken from the diagram shown in Fig.1.

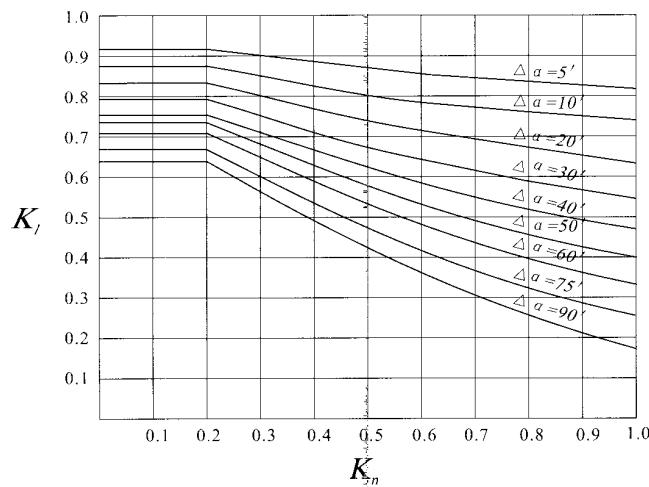


图 1 转矩修正系数

Fig.1 Torque correction factor

图中：

$\Delta \alpha$ ——被联接两轴间的轴偏角；

K_n——转速系数，用下式计算：

In diagram:

$\Delta \alpha$ -- Angle misalignment between the two shafts;

K_n -- Revolution factor, calculated below:

$$K_n = \frac{n}{[n]}$$

式中：

n——联轴器的工作转速，r/min；

[**n**]——联轴器的许用转速，r/min。

Where:

n-- Running speed of coupling, in r/min.

[**n**]-- Permissible speed of coupling, in r/min.

3.4 联轴器临界转速的验算

接中间轴或接中间套的鼓形齿式联轴器，在中间轴或中间套过长、过重及转速较高时，应验算临界转速。

3.4.1 接中间轴鼓形齿式联轴器的临界转速按下式计算：

$$n_k = 1.2 \times 10^8 \frac{d}{A^2}$$

接中间套鼓形齿式联轴器的临界转速按下式计算：

$$n_k = 1.2 \times 10^8 \frac{\sqrt{D^2 - d_1^2}}{A^2}$$

式中：

n_k ——联轴器的临界转速，r/min；

d——中间轴的外径，mm；

D——中间套的外径，mm；

d_1 ——中间套的内径，mm；

A——两端外齿齿宽中线间距离，mm。

3.4.2 鼓形齿式联轴器的工作转速与其临界转速应符合以下规定：

工作转速低于一阶临界转速：

$$n \leq 0.75n_k$$

工作转速高于一阶临界转速时：

When it is higher than first order critical speed:

$$n \geq 1.35n_k$$

3.5 鼓形齿式联轴器的齿面磨损验算

鼓形齿式联轴器的承载能力和使用寿命受齿面磨损的限制，为了避免齿面过度磨损，选定联轴器的型号和尺寸后，应验算齿面的压强：

$$p = \frac{2T_c}{Zbhd} \leq [P]$$

式中：

Z——齿数；

Where:

Z -- Number of teeth,

3.4 Check of coupling for critical speed Critical speed has to be checked for curved tooth coupling with intermediate shaft or intermediate sleeve when the intermediate shaft or sleeve is excessive long, heavy and speed is higher.

3.4.1 Critical speed of curved tooth coupling with intermediate shaft is calculated through the following formula:

Critical speed of the coupling with intermediate sleeve is Calculated as:

Where:

n_k — Critical speed of coupling, in r/min,

d-- Outer diameter of intermediate shaft, in mm,

D -- Diameter of intermediate sleeve, in mm,

d_1 -- Internal diameter of inter-sleeve, in mm,

A -- Distance between the teeth center lines of the hub with external teeth at both ends, in mm,

3.4.2 The operation speed and critical speed of coupling shall meet the following rules:

When it is lower than first order critical speed:

3.5 Check of coupling for wear of teeth face

The service life and load capacity of the curved tooth coupling is limited by the wear of tooth face To avoid excessive wear of tooth face the pressure on tooth face has to be checked after selection of the type and size of the coupling:

d—齿轮分度圆直径, mm;
 b—外齿轴套的齿宽, mm;
 h—齿轮的工作高度, mm;
 [n]—许用压强, 其值与材料及热处理、润滑条件、制造和安装精度、工作转速等因素有关, 通常可取
 $[p] = 10-15 \text{ N/mm}^2$.

3.6 联轴器齿面相对滑动速度验算
 当鼓形齿式联轴器的工作转速较高时, 按下式验算齿面轴向相对滑动速度:

d-- diameter of graduated circle, in mm,
 b -- external tooth width of the hub,
 h-- operating height of the gear, in mm,
 [P]-- permissible pressure, depending on material, heat treatment, lubrication condition, accuracy and installation, operating speed etc.
 usually taken as $[P] = 10-15 \text{ N/mm}^2$

3.6 Check of relative slipping speed on the tooth face where the operating revolution is rather high the axial relative slipping speed on tooth face is checked through the following equation:

$$V_c = \frac{nd \tan \Delta\alpha}{30000} \leq [V_{c\max}]$$

式中

V_c —联轴器齿面轴向相对滑动速度, m/s;
 $[V_{c\max}]$ —允许的最大轴向相对滑动速度, 一般可取0.12m/s。

Where:

V_c -- Axial relative slipping speed on coupling tooth face, in m/s, $[V_{c\max}]$ -- Permissible axial relative slipping max speed, normally taken as 0.12 m/s.

3.7 鼓形齿式联轴器的许用补偿量

联轴器的许用补偿量是指由于联轴器的制造和安装误差、工作时载荷变化所引起的冲击、振动、受载变形和热变形、机座不均匀的下沉以及轴承磨损等多种因素所造成的两轴线相对偏移的补偿能力, 而不是允许的安装误差。安装误差应小于所规定的许用补偿量, 一般不得大于许用补偿量的1/2—1/4。

联轴器产生相对偏移的可能情况如下图2所示。此外, 在工作状态中还可能出现同时兼有两种或三种偏移情况的综合偏移。

3.7 Permissible compensation value of coupling

The permissible compensation value of the coupling means the capability to compensate misalignment between the two shafts resulted from many factors such as machining and assembling errors, impact, vibration due to load variation during operation, deformation due to loading and heating, uneven sinking of the base frame and bearing wear etc.. It is not the permissible installation tolerance. The installation tolerance should be smaller than the specified permissible compensation value, and should not be greater than 1/2—1/4 of permissible compensation value.

The possible misalignment of the coupling is shown in Fig.2. In addition, combined misalignment which involves two or three type of displacement may appear.

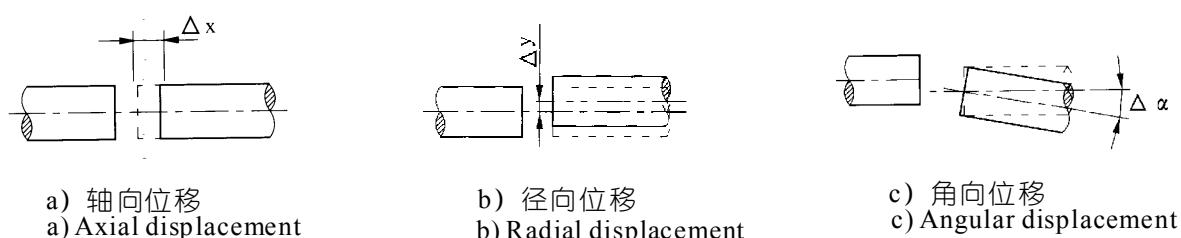


图2 轴线的相对位移
 Fig2. Relative displacement of the axes

3.7.1 当被联接两轴轴线间无径向偏移时鼓形齿式联轴器的每一对内外齿啮合的许用角向补偿量为 $\Delta \alpha = 1.5^\circ$ ，两对内外齿啮合的许用角向补偿量则为 $\Delta \alpha_{max} = 2 \times 1.5^\circ = 3^\circ$ 。

3.7.2 当被联接两轴轴线间无角向偏移时GICL和GIICL型鼓形齿式联轴器的许用径向补偿量 Δy 见附录表3和表4；接中间轴和中间套型鼓形齿式联轴器的许用径向补偿量 Δy 按下式计算，式中符号的意义同上。

3.7.1 Where there is no radial displacement between two shafts the permissible angular compensation amount for each pair of external and internal teeth meshing is $\Delta \alpha = 1.5^\circ$. For two pairs of internal and external teeth meshing $\Delta \alpha_{max} = 2 \times 1.5^\circ = 3^\circ$.

3.7.2 Where no angular displacement between two axial lines appears the permissible radial compensation amount Δy for gear coupling of type GICL and GIICL are given in A.3 and A.4. Δy for coupling with intermediate shaft and intermediate sleeve is calculated as the following, Where the symbols have the same meaning as above stated.

$$\Delta y = A \tan \Delta \alpha = 0.0262 A$$

3.7.3 鼓形齿式联轴器的许用轴向补偿量 Δx 见选型表。

3.8 其他

3.8.1 选用接中间轴和接中间套型鼓形齿式联轴器时，由于联轴器在运转中自动对中的需要，中间轴或中间套的重量值不得大于根据公称转矩计算得到的轮齿节圆处啮合圆周力的2%。

3.8.2 当选用带制动轮或制动盘鼓形齿式联轴器时，若制动器的功率大于动力机的功率，则应按制动器的功率计算理论转矩T。

3.8.3 在校核联接轴强度时，应当考虑到鼓形齿式联轴器的轮齿啮合所产生的对轴附加弯曲力矩，其值大约为 $0.1 T_{max}$ 并作用在通过轴线的平面上，其中 T_{max} 为长期作用在联轴器上的最大转矩。

3.8.4 在联轴器型号选定后，还应对轴和键的强度作校核验算。

3.7.3 Permissible axial compensation amount Δx for coupling is given in table type selection.

3.8 Others

3.8.1 When a gear coupling with intermediate shaft or intermediate sleeve is selection the weight of the intermediate shaft or intermediate sleeve should not be grater than 2% of tangential force at pitch circle meshing point of the gear and this force is obtained from calculation based on nominal torque.

3.8.2 When a gear coupling with brake wheel or brake disc is selected and if brake power is greater than the prime mover power then the theoretical torque T should be calculated based on the brake power.

3.8.3 In checking of the strength of connecting shaft the additional bending moment due to teeth meshing of the curved tooth coupling has to be considered, it is approximately $0.1 T_{max}$ and acts on the plane passing through the axis. T_{max} is the max. torque frequently imposed on the coupling

敬告用户：

随着技术进步，今后此样本中所列各系列联轴器的某些尺寸会有修正，届时恕不另行通知。

Notice consumers:

All details subject to intermediate alter actions without notice, adaption to new standards or new scientific knowledge.

4. GICL型—基本型鼓形齿式联轴器的结构型式、参数和尺寸

4. Type GICL-Curved tooth coupling and its construction, parameters and dimensions

GICL基本型鼓形齿式联轴器（符合Q/WL003-2007，参照JB/T88543-2001），其特征是内齿圈较宽，能补偿较大的轴线偏移，适用于联接水平两同轴线轴系传动。其结构型式见图3，基本参数和主要尺寸见表3。

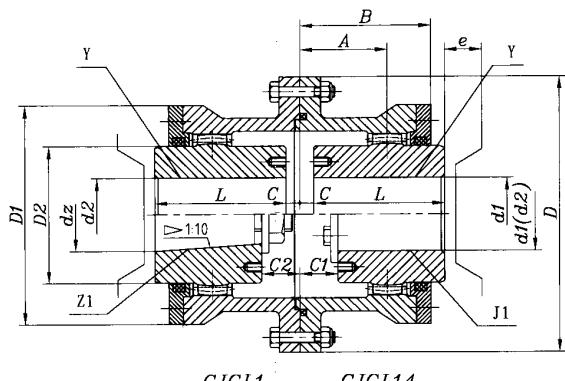
GICL基本型鼓形齿式联轴器允许正、反方向回转，可将任一侧外齿轴套作为主动输入端，传递公称转矩为630~2800000 Nm。

为增强润滑密封效果、减少零件数量，提高运行可靠性，特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

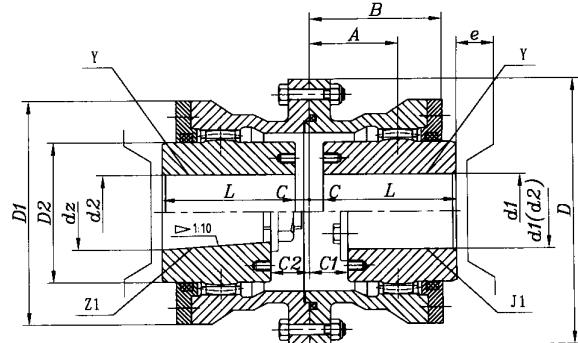
Curved tooth coupling type GICL (In compliance with Q/WL003-2007, Reference JB/T8854.3-2001) has rather wide internal gear ring to compensate fairly large axes displacement and is suitable for connecting two horizontal co-axial shaft. Its construction is shown in Fig.3, the basic parameters and main dimensions are given in Table 3.

Coupling type GICL may be rotated in two directions and any one side external geared hub can be used as the driven input end. The torque transmitted can be in the range of 63 0 through 2,800,000N. m

To increase lubrication and sealing effects, to reduce number of parts and to enhance operating reliability it is recommended to use integrated structure of sealing end cover with internal gearing



GICL1 - GICL14



GICL15 - GICL30

图3 GICL型鼓形齿式联轴器结构型式

Fig.3 Construction of the curved tooth coupling GICL

Y—Y型轴孔 J1—J1型轴孔 Z1—Z1型轴孔

Y -- Bore Type Y J1 --Bore Type J1, Z1--Bore Type Z1

表3 GICL型鼓形齿式联轴器的基本参数和主要尺寸 (符合Q/WL003-2007, 参照JB/T8854.3-2001)

Tab.3 Parameters and Dimensions of the Couplings Type GICL (In compliance with Q/WL003-2007, ref. JB/T8854.3-2001) mm

型号 Type	公称转矩 Tn (N·m)	许用转速 Speed [n] (r/min)	轴孔直径 Bore d1,d2,dz		轴孔长度 Length of bore L		D	D1	D2	A	B	C	C1	C2	e	润滑脂用量 Qty of Lub V (mL)	质量 Mass m (kg)	转动惯量 Inertia I (kg · m ²)
			从 Form	到 To	Y	J ₁ Z ₁												
GICL1	630	4000	16	19	42	-	125	85	55	30	58	20	-	-	30	55	5.9	0.01
			20	24	52	38						10						
			25	28	62	44						2.5						
			30	38	82	60						15						
GICL2	1120	4000	25	28	62	44	144	105	65	37	66	10.5	-	29	30	100	9.7	0.02
			30	38	82	60						2.5						
			40	45	112	84						13.5						
GICL3	2240	4000	30	38	82	60	174	125	85	43	75	24.5	25	30	140	17.2	0.05	
			40	55	112	84						3						
			60		142	107						17						
GICL4	3550	3600	32	38	82	60	196	148	105	47	82	14	3.7	32	30	170	24.9	0.09
			40	55	112	84						3						
			60	70	142	107						17						
GICL5	5000	3300	40	55	112	84	224	168	120	51	90	25	28	30	270	38	0.17	
			60	75	142	107						3						
			80	85	172	132						22						
GICL6	7100	3000	48	55	112	84	241	188	135	56	97	6	35	35	30	380	48.2	0.27
			60	75	142	107						4						
			80	90	172	132						22						
GICL7	10000	2680	60	75	142	107	265	210	150	62	104	4	25	35	30	570	68.9	0.45
			80	95	172	132						22						
			100	105	212	167						48						
GICL8	14000	2500	65	75	142	107	285	232	165	63	107	5	35	30	660	83.3	0.65	
			80	95	172	132						22						
			100	115	212	167						48						
GICL9	18000	2350	70	75	142	107	314	247	190	66	112	10	45	45	30	700	110	1.04
			80	95	172	132						5						
			100	120	212	167						22						
GICL10	31500	2150	80	95	172	132	346	278	210	73	121	5	43	43	30	900	157	1.88
			100	125	212	167						29						
			130	140	252	202						54						
GICL11	40000	1880	100	120	212	167	385	317	250	86	143	6	49	40	1200	217	3.28	
			130	150	252	202						29						
			160	170	302	242						64						
GICL12	56000	1680	100	120	212	167	442	363	280	94	157	6	57	40	2000	305	5.08	
			130	150	252	202						29						
			160	180	302	242						68						

5. GICLZ型—接中间轴鼓形齿式联轴器的结构型式、参数和尺寸

5. Type GICLZ-Curved tooth coupling and its construction, parameters and dimensions

GICLZ型接中间轴鼓形齿式联轴器（符合

Q/WL003-2007，参照JB/T8854.3 – 2001），内齿圈较宽，能补偿较大的轴线偏移，适用于联接长距离传动、联接的轴向尺寸较大的水平两同轴线轴系传动其结构型式见图4，基本参数和主要尺寸见表4。

GICLZ型接中间轴鼓形齿式联轴器允许正、反方向回转，可将任一侧作为主动输入端，传递公称转矩为630 – 2800000Nm。一般情况下，应成对使用，外齿轴套端与中间轴联接，两端半联轴器则分别与工作机轴和动力机轴联接。

由于联轴器在运转中自动对中的需要，中间轴的重量值不得大于根据公称转矩计算得到的轮齿节圆处啮合圆周力的2%; 在中间轴过长、过重及转速较高时，应验算临界转速。

为增强润滑密封效果、减少零件数量，提高运行可靠性，建议选用密封端盖与内齿圈作成一体的整体结构型式。

Curved tooth coupling Type GICLZ with intermediate shaft (in compliance with Q/WL003-2007, ref JB/T8854.3-1999) has rather wider internal gear ring and is capable of compensating relatively large axes displacement and suitable for long distance transmission to couple two horizontal co-axial shaft system with large axial dimensions. Its main dimensions are given in Table4.

Gear coupling of Type GICLZ allows rotation in two directions. Any one side may be used as driven input end. Nominal transmitted torque lies in a range of 63 0 through 2,800,000Nm. Normally it is used in pairs, the hub with external teeth is connected with intermediate shaft and the two coupling halves are connected with the shafts of working machine and of prime mover respectively.

The weight of the intermediate shaft should not be greater than 2% of tangential force at the meshing points. This is due to requirement for self alignment of the coupling during operation. Critical revolution should be checked in the case of excessive length and weight of intermediate shaft and under high revolution.

To increase lubrication sealing effect, to reduce number of parts and enhance operation reliability it is recommended to use integrated sealing end cover with internal gearing.

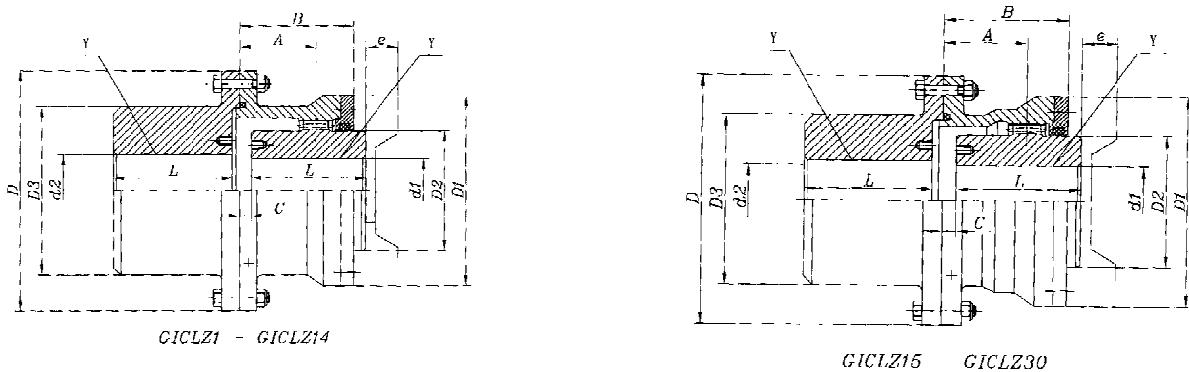


图4 GICLZ型鼓形齿式联轴器结构型式

Fig. 4 Construction of the curved tooth coupling Type GICLZ

Y—Y型轴孔

Y—Bore Type Y

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表4 GICLZ型鼓形齿式联轴器的基本参数和主要尺寸(符合Q/WL003-2007, 参照JB/T8854.3-2001)

Tab.4 Parameters and Dimensions of the Couplings Type GICLZ(in compliance with Q/WL003-2007)

mm

型号 Type	公称转矩 Torque Tn (N·m)	许用转速 Speed [n] (r/min)	轴孔直径 Bore d1,d2,dz		轴孔长度 Length of bore L	D	D1	D2	D3	A	B	C	e	润滑脂 用量 Qty of Lub V (mL)	质量 Mass m (kg)	转动惯量 Inertia I (kg·m²)
			从 Form	到 To												
GICLZ1	630	4000	16	19	42	125	85	55	80	30	58	20	30	5.4	0.008	
			20	24	52											
			25	28	62											
			30	38	82											
			40	50	112											
GICLZ2	1120	4000	25	28	62	144	105	65	95	37	66	10.5	30	60	9.2	0.018
			30	38	82											
			45	50	112											
			55	65	142											
GICLZ3	2240	4000	30	38	82	174	125	85	115	43	75	3	30	80	16.4	0.042
			40	55	112											
			60	75	142											
GICLZ4	3550	3600	32	38	82	196	148	105	130	47	82	14	30	90	22.7	0.076
			40	55	112											
			70	75	142											
			80	85	172											
GICLZ5	5000	3300	40	55	112	224	168	120	150	51	90	3	30	140	36.2	0.148
			60	75	142											
			85	95	172											
GICLZ6	7100	3000	48	55	112	241	188	135	170	56	97	6	30	200	46.2	0.238
			60	75	142											
			90	95	172											
			100	110	212											
GICLZ7	10000	2680	60	75	142	265	210	150	190	62	104	4	30	290	68.4	0.425
			80	95	172											
			105	120	212											
GICLZ8	14000	2500	65	75	142	285	232	165	210	63	107	5	30	350	81.1	0.605
			80	95	172											
			100	115	212											
			130	140	252											
GICLZ9	18000	2350	70	75	142	314	247	190	225	66	112	10	30	370	100.1	0.935
			80	95	172											
			100	120	212											
			130	150	252											
GICLZ10	31500	2150	80	95	172	346	278	210	250	73	121	5	30	500	147.1	1.65
			100	125	212											
			140	150	252											
			160	165	302											
GICLZ11	40000	1880	100	120	212	385	317	250	285	86	143	6	40	650	206.3	2.95
			130	150	252											
			170	190	302											
			-	120	212											
GICLZ12	56000	1680	130	150	252	442	363	280	325	94	157	6	40	1100	284.5	5.26
			160	180	302											
			190	210	352											

6. GICLT型-接中间套鼓形齿式联轴器的结构型式、参数和尺寸

GICLT型宽型接中间套鼓形齿式联轴器（符合Q / WL 003 – 2007），内齿圈较宽，能补偿较大的轴线偏移，适用于联接长距离传动、联接的轴向尺寸较大时的水平两同轴线轴系传动。其结构型式见图5，基本参数和主要尺寸见表5。

GICLT型接中间套鼓形齿式联轴器允许正、反方向回转，也可将任一侧作为主动输入端，传递公称转矩为630 – 2800000Nm。

由于联轴器在运转中自动对中的需要，中间套的重量值不得大于根据公称转矩计算得到的轮齿节圆处啮合圆周力的2%；在中间套过长、过重及转速较高时，应验算临界转速。

为了增强润滑密封效果、减少零件数量，提高运行可靠性，特别建议选用密封端盖与内齿圈作成一体的整体结构型式。中间套的质量和转动惯量见表15。

6. Type giclt-curved tooth coupling and its construction, parameters and dimensions

Curved tooth coupling with intermediate sleeve (in compliance with Q/WL01.01-2002) has rather wide internal gear rings and is able to compensate relatively large axes displacement. It is suitable for long distance transmission and to connect two horizontal co-axial shaft system with large axial dimensions. Construction is shown in Fig.5. Basic parameters and main dimensions are given in Table.5.

Curved tooth coupling with intermediate sleeve type GICLT may be rotated in two directions. Any one side may be used as driven input end. Transmitted nominal torque ranges from 630 to 2,800,00Nm.

The weight of intermediate sleeve should not be greater than 2% of tangential force at the meshing point on pitch circle. The tangential force is obtained by calculation based on nominal torque. This is because of requirement for self alignment during coupling running. In case of excessive length and weight of the intermediate sleeve and high speed the critical revolution must be checked.

To increase lubrication sealing effect, to reduce number of parts and increase operation reliability it is recommended to use integrated construction of sealing end cover with internal gearing.

The mass and moment of inertia of intermediate sleeve are given in Table 15.

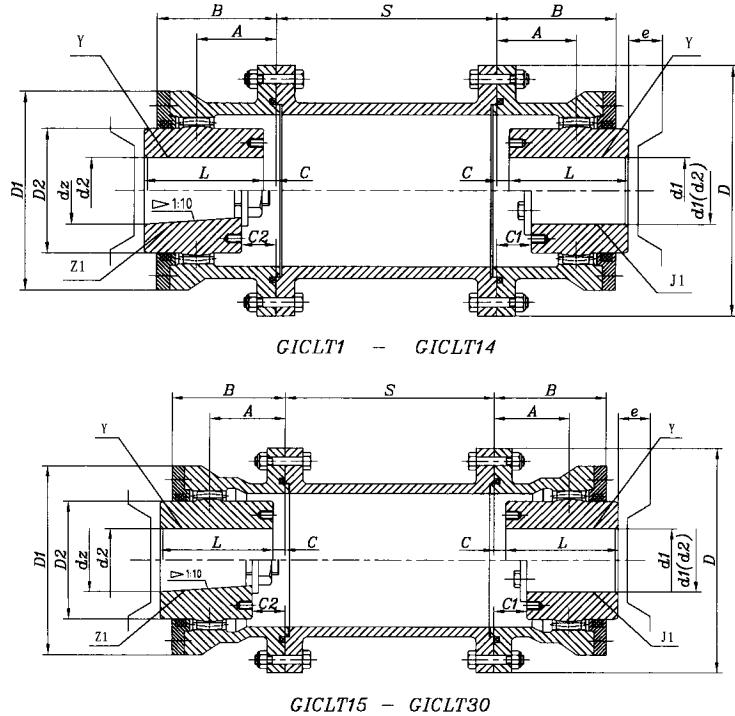


图5 GICLT型鼓形齿式联轴器结构型式
Figs Construction of the curved tooth coupling GICLT
Y—Y型轴孔 J1—J1型轴孔 Z1—Z1型轴孔
Y—bore type Y J—bore type J Z—bore type Z1

7. GGCL型—套管型双鼓形齿式联轴器的结构型式、参数和尺寸

7.type GGCL-Curved tooth coupling and its construction,parameters and dimensions

GGCL 型套管型双鼓形齿式联轴器(符合Q

/ WL 003-2007) 内齿圈较宽结构轻巧, 转动惯量小, 能补偿较大的轴线偏移, 适用于联接水平两同轴线传动轴系、中小载荷高速场合。其结构型式见图6、基本参数和主要尺寸见表6。

GGCL型套管型双鼓形齿式联轴器允许正、反方向回转, 可将任一侧外齿轴套作为主动输入端, 传递公称转矩为630-112000N · m

Curved tooth coupling type

GGCL(incompliance with Q / WL01.02—2002)has rather wide internal gear ring, light structure and low moment of inertia and is able to compensate large axes misalignment . It is suitable for connection of two horizontal CO-axial transmission shaft systems with low and medium load and high speed. Construction is shown in Fig6 and basic parameters and main dimensions are given in Table 6.

Coupling type GGCL may rotate in both directions and any side external gear—hub can be used as the driven input end. Transmitted nominal torque lies. In the range from 630 to 11200 N · m.

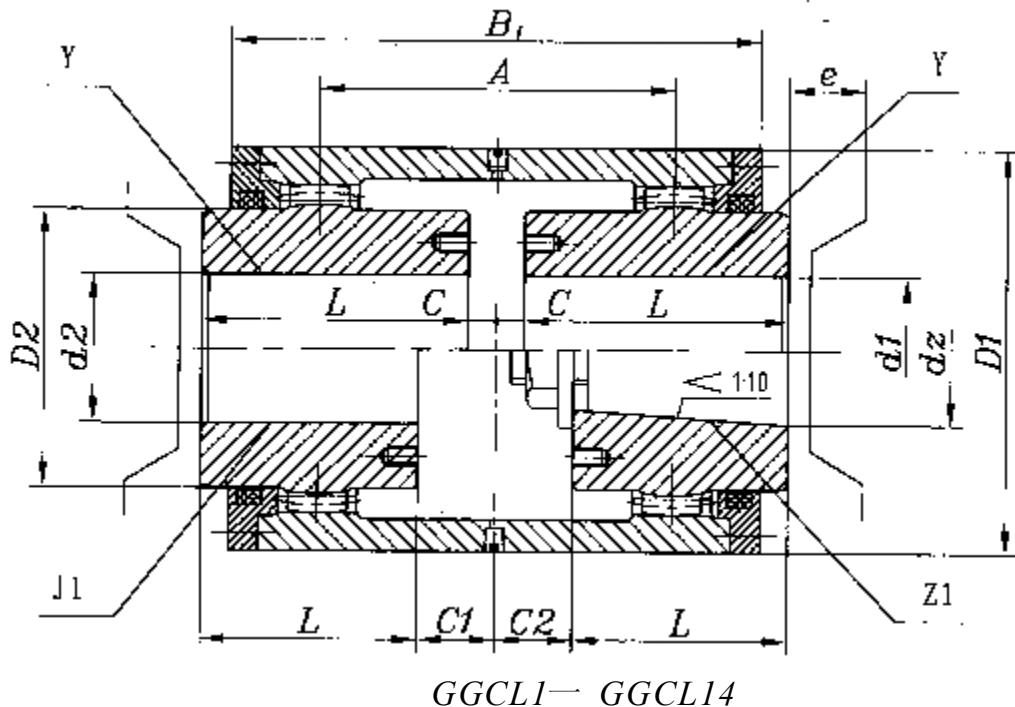


图6 GGCL型鼓形齿式联轴器结构型式

Fig.6 Construction of the curved tooth coupling Type GGCL

Y—Y型轴孔 J₁—J₁型轴孔 Z₁—Z₁型轴孔

Y—Bore Type Y J₁—Bore Type J₁ Z₁—Bore Type Z₁

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表6 GGCL型套管型双鼓形齿式联轴器的基本参数和主要尺寸 (符合Q/WL003-2007)
 Tab.6 Parameters and Dimensions of the Coupling Type GGCL(in compliance with Q/WL003-2007) mm

型号 Type	公称转速 Torque T n (N•m)	许用转速 Speed [n] (r/min)	轴孔直径 Bore d1 d2 d3		轴孔长度 Length of bore L		D ₁	D ₂	A	B ₁	C	C ₁	C ₂	e	润滑脂 用量 Q'ty of Lub V (mL)	质量 Mass m (kg)	转动惯量 Inertia I (kg m)
			从 Form	到 To	Y	J ₁ Z ₁											
GGCL1	630	4000	16	19	42	-	100	55	60	116	20	30	55	5.0	0.01		
			20	24	52	38					10						
			25	28	62	44					2.5						
			30	38	82	60					15	22					
GGCL2	1120	4000	25	28	62	44	120	65	74	132	10.5	29	30	100	8.3	0.02	
			30	38	82	60					2.5	12.5	30				
			40	45	112	84					13.5	28					
GGCL3	2240	4000	30	38	82	60	140	85	86	150	24.5	25	30	140	14.6	0.04	
			40	35	112	84					3	28					
			60	-	142	107					17	35					
GGCL4	3550	3600	32	38	82	60	162	105	94	164	14	37	32	30	170	21.2	0.07
			40	55	112	84					3	17	28				
			60	70	142	107					35						
GGCL5	5000	3000	40	56	112	84	190	120	102	180	25	28	30	270	32.3	0.14	
			60	75	142	107					3	20	35				
			80	85	172	132					22	43					
GGCL6	7100	3000	48	56	112	84	205	135	112	194	6	35	35	30	380	41.2	0.26
			60	75	142	107					4	20	35				
			80	90	172	132					22	43					
GGCL7	10000	2680	60	75	142	107	230	150	124	208	4	25	35	30	570	58.6	0.36
			80	95	172	132					22	43					
			100	105	212	167					48						
GGCL8	14000	2500	65	75	142	107	255	165	126	214	5	35	35	30	660	70.8	0.52
			80	95	172	132					22	43					
			100	115	212	167					48						
GGCL9	18000	2350	70	75	142	107	272	190	132	224	10	45	45	30	700	93.5	0.83
			80	95	172	132					5	22	43				
			100	120	212	167					49						
GGCL10	31500	2150	80	95	175	132	300	210	146	242	5	43	43	30	900	133.5	1.50
			100	125	212	167					22	49					
			130	140	252	202					29	54					
GGCL11	40000	1880	100	120	212	167	342	250	172	286	6	29	49	40	1200	184.5	2.62
			130	150	252	202					29	54					
			160	170	302	242					64						
GGCL12	56000	1680	100	120	212	167	384	280	188	214	6	57	58	40	2000	259.3	4.06
			130	150	252	202					29	55					
			160	180	302	242					68						
GGCL13	80000	1550	140	150	252	202	430	310	218	350	7	54	57	40	3000	356.2	8.05
			160	180	302	242					32	70					
			190	210	352	282					8	42	70				
GGCL14	112000	1500	160	180	302	242	465	350	220	358	8	32	80	40	4500	504.9	13.4
			190	220	282	282					32	80					

注：1. 联轴器质量和转动惯量是按各型号中 J₁ 型轴伸计算的近似值；
 2. e为更换密封所需尺寸；

Note:1. Mass & Inertia approximate based on min.bore and max.Lenth;
 2."e" is used for seal replacement.

8. NGICLZ型-带制动轮鼓形齿式联轴器的结构型式、参数和尺寸

8. Type NGICLZ-Curved tooth coupling and its construction,parameters and dimensions

NGICLZ型接中间轴带制动轮鼓形齿式联轴器(符合 Q / W L 003--2007)，内齿圈较宽，能补偿较大的轴线偏移，适用于瓦块式制动器配套的场合，允许正反方向旋转，其结构型式见图7,基本参数和主要尺寸表7。

一般情况下,NGICLZ型联轴器应与GICLZ型联轴器配套使用，外齿轴套端与中间轴联接，两端半联轴器则分别与工作机轴和动力机轴联接。

由于制动轮与半联轴器连接在一起,制动轮的重量以及工作制动负荷与振动完全由半联轴器承受,改善了制动时鼓形齿的啮合性能,因此特别适用于重载场合,传递公称转矩为630Nm~112kNm。

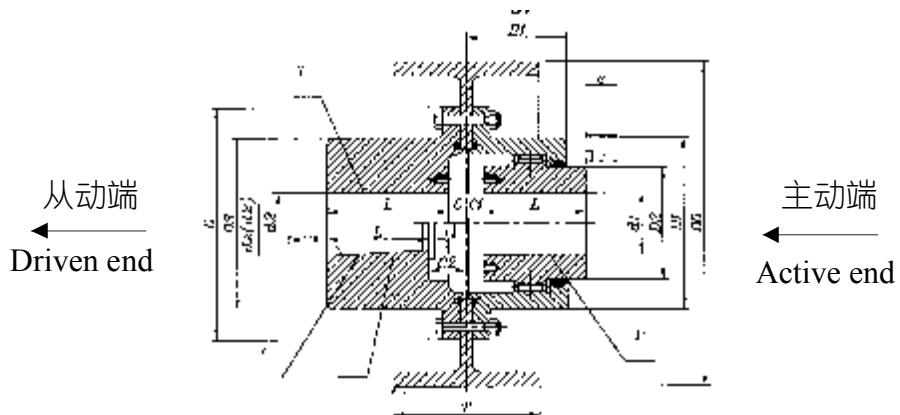
不同规格制动轮的主要参数列于表16中。

Curved tooth coupling with intermediate shaft and brake wheel NGICLZ (in compliance with Q/WL 003-2007), It has wide inner gear ring, can compensate large axial displacement, suit for working place with slipper brake, and can be used in forward and reverse rotatory , Its construction is shown in Fig7 and its basic parameters basic parameters and dimensions are given in Table7.

Normally, coupling type NGICLZ should be used in pairs with type GICLZ. External toothed hub is connected with intermediate shaft, and the two coupling halves at both ends are connected with shafts of working machine and prime mover respectively.

Since the brake wheel is connected with the coupling half anthus the weight of brake wheel and braking load and vibration is borne by the coupling half that improves meshing performance of the crowned teeth during braking. Therefore,it is particularly suitable for heavy load duty application, the nominal transferring torque is 630Nm~112kNm.

Parameters of brake wheels, of various sizes are given in table 16.



NGICLZ1 - NGICLZ1

图7 NGICLZ 型鼓形齿式联器结构式
Fig.7 construction of the curved tooth coupling NGICLZ

Y — Y型轴孔 Y — Bore Type Y	J ₁ — J ₁ 型轴孔 J ₁ — Bore Type Z	Z — Z型轴孔 ZBore Type Z	Z ₁ — Z ₁ 型轴孔 Z ₁ — Bore Type Z ₁
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9. PGCLZ型-带制动盘鼓形齿式联轴器的结构型式、参数和尺寸

9. Type PGCLZ-Curved tooth coupling and its construction,parameters and dimensions

PGCLZ型接中间轴带制动盘鼓形齿式联轴器（符合Q / WL003-2007），内齿圈较宽，能补偿较大的轴线偏移，适用于与盘式制动器配套的场合，允许正、反方向回转。其结构型式见图8，基本参数和主要尺寸见表8。

一般情况下，PGCLZ 型联轴器应与GICLZ型联轴器配对使用，外齿轴套端与中间轴联接，两端半联轴器则分别与工作机轴和动力机轴联接。由于制动盘与半联轴器连接在一起，制动盘的重量以及工作制动负荷与振动完全由半联轴器承受，改善了制动时鼓形齿的啮合性能，因此适用于重载场合，传递公称转矩为630-112000 N · m。

为增强润滑密封效果、减少零件数量，提高运行可靠性，特别建议：PGCLZ型带制动盘鼓形齿式联轴器选用密封端盖与内齿圈作成一体的整体结构型式。

当配用不同规格的制动盘时，制动盘的有关参数见表17。

Curved tooth coupling with intermediate shaft and brake disc PGCLZ(in compliance with Q/WL 01.05—2002) has relatively wide internal geared ring and can rather large axes misalignment. It is suitable for application in conjunction with disc brake for rotation in both directions. Its construction is shown in Fig.8 and basic parameters and main dimensions are given in Table 8.

Normally, coupling type PGCLZ should be used in conjunction with coupling of type GICLZ. External toothed hub is connected with intermediate shaft, the coupling halves at both ends are connected with shaft of the working machine and the prime mover respectively. Since the brake disc is connected with the coupling half the weight of brake disc , braking load and vibration are borne by the coupling half and thus the meshing performance of crowned teeth during braking is improved. Therefore , it is suitable for application with heavy load , Transmitted nominal torque ranges from 630 to 112,00 N · m.

To increase lubrication sealing effect and operation reliability , to reduce parts number it is especially recommended that for gear coupling type PGCLZ with brake disc the integrated construction of sealing end cover with internal gear ring should be used.

Parameters of brake disc of different sizes are

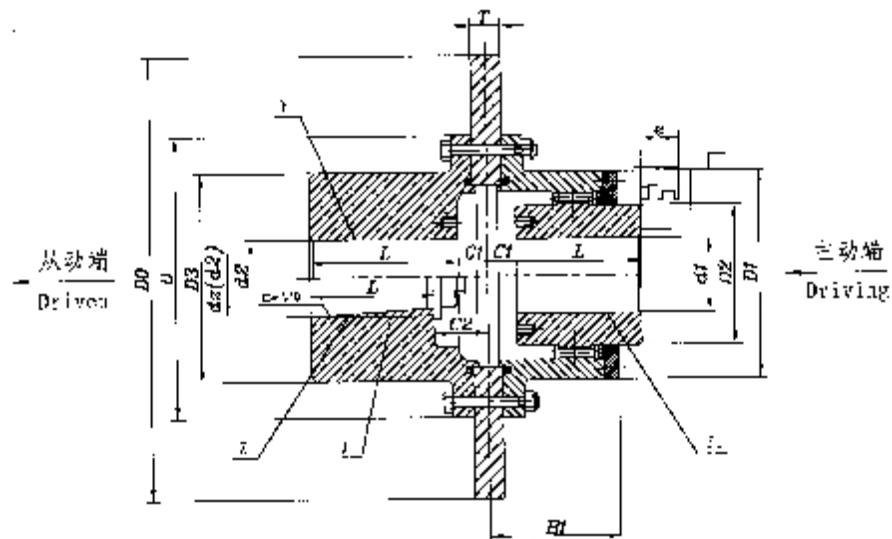


图8 PGCLZ型鼓形齿式联轴器结构型式

Figure 8 Construction of the curved tooth coupling PGCLZ

Y—Y型轴孔 J₁—J₁型轴孔 Z—Z型轴孔

Y-Bore TypeY J₁-BoreTypeJ₁ Z-BoreTypeZ

10. PGCLK型-带制动盘鼓形齿式联轴器的结构型式、参数和尺寸

10. TYPE PGCLK-Curved tooth coupling and its construction,parameters and dimensions

PGCLK 型带制动盘鼓形齿式联轴器(符合 Q / W L003-2007), 适用于与盘式制动器配套的场合, 允许正、反方向回转。其特点是: 双鼓形齿结构, 能补偿较大的轴线偏移; 制动盘布置在联轴器的被动端, 制动盘的重量以及工作制动负荷与振动完全由半联轴器承受, 从而改善了制动时鼓形齿的啮合性能; 最大制动力矩不受联轴器结构强度的限制, 安全可靠; 结构紧凑, 制动盘散热条件好; 插入式制动盘可快速更换而不必移开设备。

因此特别适用于负荷、转速和方向变化频繁的重载场合, 传递公称转矩为630-112000Nm。其结构型式见图9, 基本参数和主要尺寸见表9。当配用不同规格的制动盘时, 制动盘的有关参数见表17。

Coupling PGCLK with brake disc (in compliance with Q/WL003 — 2007) has the same application as above mentioned . It has features : double dcurred teeth to compensate rather large axes.displacement;the brake disc is located at the idle end of coupling and the weight of brake disc and braking load and vibration are fully borne by the coupling half and thus meshing performance of curved teeth during braking is improved; the max, braking torque is not limited by strength of coupling structure, and thus. it is safe and reliable; it has compact construction and good heat dissipation condition of brake disc; the brake disc is of inserted type that makes quick replacement possible without removing anything.

It Is particularly suitable for application with frequently varied load, speed and direction and heavy load. Transmitted nominal torque is in the range of 630-11200N · m. Its construction is shown in Fig.9 and basic parameters and major dimensions are given in Tab.9. Parameters of brake disc of different sizes are given in Tab.17.

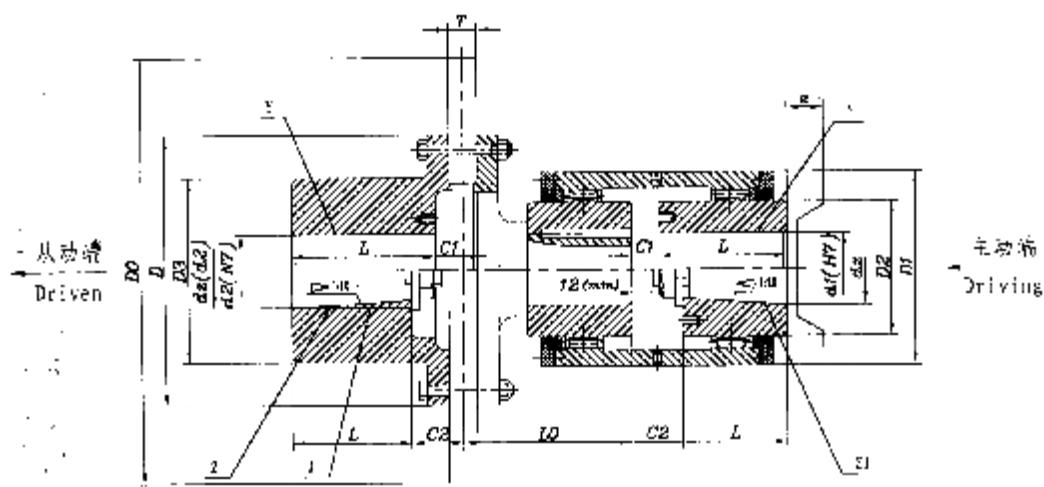


图9 PGCLK型鼓形齿式联轴器结构型式

Figure9 Construction of the Curved tooth coupling PGCLK
 Y—Y型轴孔 J—J型轴孔 Z—Z型轴孔 Zi—Zi型轴孔
 Y-Bore Type Y J-Bore Type J Z-Bore Type Z Zi-Bore Type Zi

11. GIICL型-基本型鼓形齿式联轴器的结构型式、参数和尺寸

11. Type GIICL-Curved tooth coupling and its construction,parameters and dimensions

GIICL基本型鼓形齿式联轴器齿间距小，允许相对径向位移小，结构紧凑，转动惯量小。其结构型式见图10，基本参数和主要尺寸见表10。

GIICL型鼓形齿式联轴器允许正、反方向回转，可将任一侧外齿轴套作为主动输入端，传递公称转矩为400~5000000Nm。

为增强润滑密封效果、减少零件数量，提高运行可靠性，特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

Flexible gear coupling GIICL has small teeth pitch and allows small radial relative displacement, it has compact structure and low moment of inertia. Construction of Type GIICL is shown in Fig.10 and parameters and dimensions are given in Table.10.

Gear coupling GIICL allows rotation in two directions and any one side external toothed hub may be used as the driven input end. Transmitted nominal torque ranges from 400 to 5,000,000 Nm.

To enhance lubrication sealing effect, to reduce number of parts and increase operation reliability it is specially recommended to use integrated sealing end cover with inner gear.

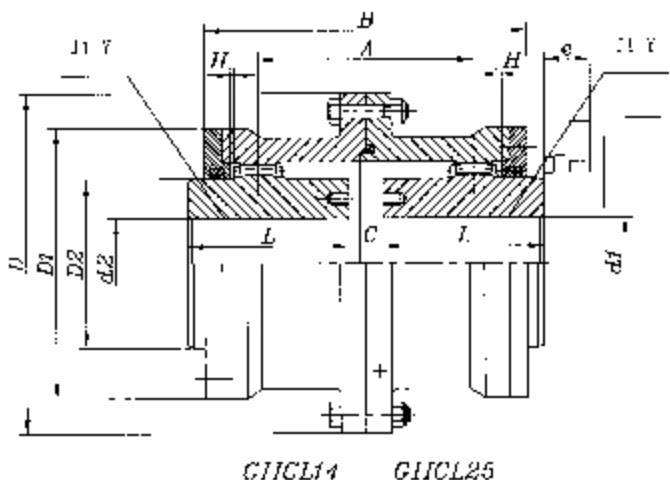
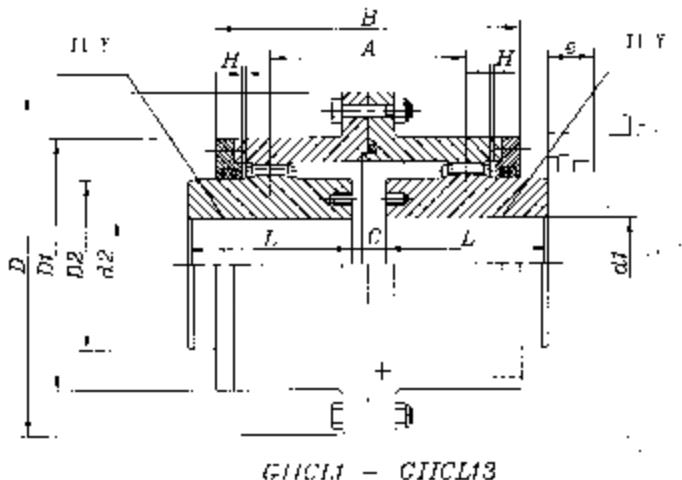


图10 GIICL型鼓形齿式联轴器结构型式

Fig.10 Construction of the Curved tooth coupling GIICL

Y-Y型轴孔 J-J型轴孔

Y-Bore TypeY J-Bore TypeJ

12. GCLD型-接电机轴伸鼓形齿式联轴器的结构型式、参数和尺寸

12. Type GCLD-Curved tooth coupling and its construction,parameters and dimensions

GCLD型接电机轴伸鼓形齿式联轴器采用GIICL基本型的齿形啮合参数，齿间距小，允许相对径向位移小，结构紧凑，转动惯量小，适用于与电机配套的场合。其结构型式见图11，基本参数和主要尺寸见表11。

GCLD型接电机轴伸鼓形齿式联轴器允许正、反方向回转，可将任一侧外齿轴套作为主动输入端，传递公称转矩为1120-50000N·m。

为增强润滑密封效果、减少零件数量，提高运行可靠性，特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

Curved tooth coupling connected with electric motor shaft extension Type GCLD has the teeth meshing parameters of type GIICL with small pitch and small permissible radial displacement, compact structure, low moment of inertia and is suitable for using in conjunction with electric motor. Its construction is shown in Fig.11 and basic parameters and dimensions are given in Table 11.

Type GCLD may rotate in two directions and any one side external toothed hub may be used as input end. Transmitted nominal torque ranges from 1120 to 50000 Nm.

To enhance lubrication sealing effect, to reduce number of parts and increase operation reliability it is specially recommended to use integrated sealed end cover with internal gear ring.

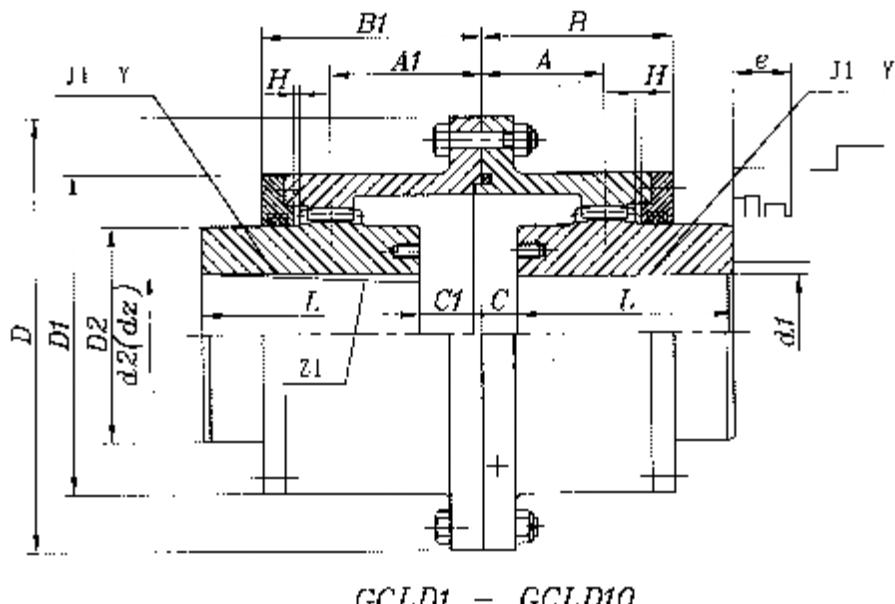


图11 GCLD型鼓形齿式联轴器结构型式
Fig.11 Construction of the curved tooth coupling GCLD
Y-Y型轴孔 J₁-J₁型轴孔
Y-BoreType Y J₁-BoreType J₁

13.GIICLZ型-接中间轴鼓形式联轴器的结构型式、参数和尺寸

13.Type GIICLZ-Curved tooth coupling and its construction,parameters and dimensions

GIICLZ型接中间轴鼓形齿式联轴器,齿间距小,允许相对径向位移小,结构紧凑,适用联接长距离传动的水平两同轴线轴系传动。其结构型式见图12, 基本参数和主要尺寸见表12。

GIICLZ型接中间轴鼓形齿式联轴器允许正、反方向回转,也可将任一侧作为主动输入端,传递公称转矩为400-5000000N·m。一般情况下,应成对使用,外齿轴套端与中间轴联接,两端半联轴器则分别与工作机轴和动力机轴联接。

由于联轴器在运转中自动对中的需要,中间轴的重量值不得大于根据公称转矩计算得到的轮齿节圆处啮合圆周力的2%;在中间轴过长、过重及转速较高时,应验算临界转速。

为增强润滑密封效果、减少零件数量,提高运行可靠性,特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

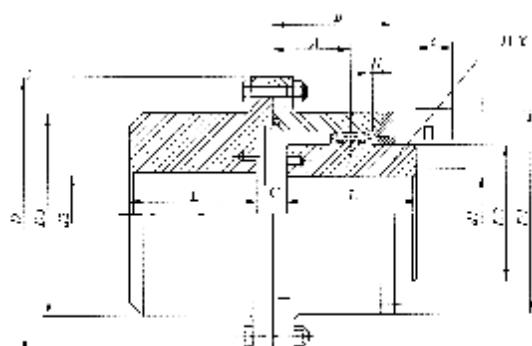
Curved tooth coupling with intermediate shaft

GIICLZ has small teeth pitch and compact suitable for long distance axial transmission. Its construction is shown in Fig.12 and basic parameters and dimensions are given in Table 12.

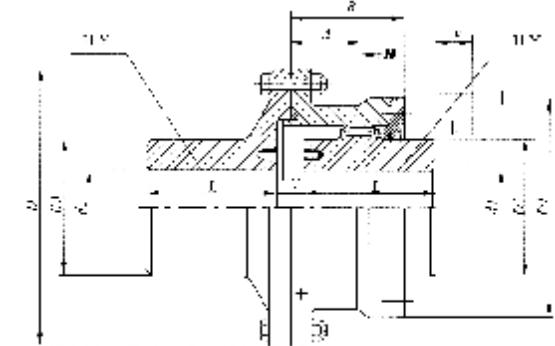
Coupling Type GIICLZ allows to rotate in two directions and any one side may be used as input end. Trans mitted nominal torque ranges from 400 to 5,000, 000 Nm. Normally it should be used in pairs, the external toothed hub is connected with intermediate shaft and the two coupling halves at both ends are connected with shafts of the working machine and prime mover respectively.

The weight of intermediate shaft should not be greater than 2% of the tangential force at meshing point on pitch circle. In case of excessive long and heavy intermediate shaft and under high speed critical speed should be checked.

To enhance lubrication sealing effect, to reduce number of parts and increase operation reliability it is recommended to use integrated sealed end Cover with internal geared ring.



GIICLZ1 - GIICLZ13



GIICLZ14 - GIICLZ25

图12 GIICLZ型鼓形齿式联轴器结构型式

Fig.12 Construction of the carved tooth coupling e GIICLZ

Y-Y型轴孔	J ₁ -J ₁ 型轴孔
Y-Bore Type Y	J ₁ -Bore Type J ₁

14. GIICLT型—接中间套鼓形齿式联轴器的结构型式、参数和尺寸

14. Type GIICLT-Curved tooth coupling and its construction,parameters and dimensions

GIICLT型窄型接中间套鼓形齿式联轴器, 齿间距小, 允许相对径向位移小, 结构紧凑, 适用范围同上。其结构型式见图13, 基本参数和主要尺寸见表13。

GIICLT型接中间套鼓形齿式联轴器允许正、反方向回转, 也可将任一侧作为主动输入端, 传递公称转矩为400-5000000N·m。一般情况下, 应成对使用, 内齿圈, 与中间套联接, 两端外齿轴套则分别与工作机轴和动力机轴联接。

由于联轴器在运转中自动对中的需要中间套的重量值不得大于根据公称转矩计算得到的轮齿节圆处啮合圆周力的2%；在中间套过长、过重及转速较高时, 应验算临界转速。

为增强润滑密封效果、减少零件数量, 提高运行可靠性, 特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

Curved tooth coupling with intermediate sleeve GIICLT has small teeth pitch and allows small relative radial displacement. It has compact structure and is suitable for the application as above mentioned. Its construction is shown in Fig.13 and basic parameters and dimensions are given in Table 13.

Curved tooth coupling with intermediate sleeve GIICLT may be rotated in either direction and any one side may be used as input end. Transmitted nominal torque ranges from 400 to 5,000,000 Nm. Normally it will be used in pairs. The internal gear ring is connected with intermediate sleeve and the external tooth hubs at both ends are connected with the shafts of machine and prime mover respectively.

The weight of intermediate sleeve should not be greater than 2% of tangential force at meshing point on pitch circle due to requirement for self-alignment of coupling during running. In case of excessive long and heavy intermediate sleeve and high speed critical revolution shall be checked.

To enhance lubrication sealing effect. To reduce number of parts and increase running reliability it is specially recommended recommended to use integrated structure of sealed end cover with internal gearing.

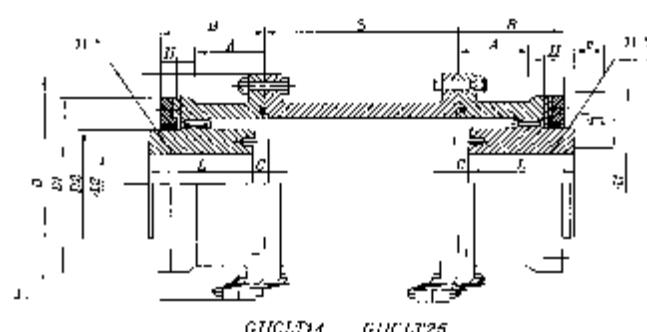
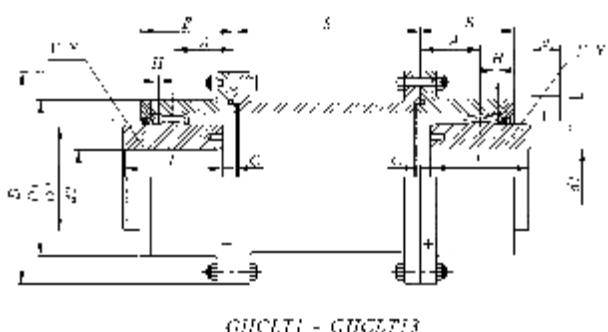


图 13 GIICLT型鼓形齿式联轴器结构型式
Fig.13 construction of the curved tooth coupling GIICLT

Y-Y型轴孔 J-J型轴孔

Y-BoreType Y J-BoreType J

16. NGIICLZ型-带制动轮鼓形齿联轴器的结构式、参数和尺寸

16. Type NGIICLZ-Curved tooth coupling and its construction, parameters and dimensions

NGIICLZ型接中间轴带制动轮鼓形齿式联轴器, 齿间距小, 允许相对径向位移小, 结构紧凑, 转动惯量小, 适用于与瓦块式制动器配套的场合, 允许正、反方向回转, 其结构型式见图14, 基本参数和主要尺寸见表14。

一般情况下, NGIICLZ型联轴器应与GIICLZ型联轴器配套使用, 外齿轴套端与中间轴联接, 两端半联轴器则分别与工作机轴和动力机轴联接, 由于制动轮与半联轴器连接在一起, 制动轮的重量以及工作制动负荷与振动完全由半联轴器承受, 改善了制动时鼓形齿的啮合性能, 因此特别适用于重载场合, 传递公称转矩为400-112000Nm,

为增强润滑密封效果、减少零件数量, 提高运行可靠性, 特别建议选用密封端盖与内齿圈作成一体的整体结构型式。

不同规格制动轮的有关参数见表16。

Gear coupling connected with intermediate shaft and brake wheel type NGIICLZ has small teeth pitch and allows small radial relative displacement. It has compact structure and low moment of inertia and is suitable for application with brake of brake shoe type. It allows to rotate in two directions. Its construction is shown in Fig.10 and essential parameters and dimensions are given in Table 16.

Normally coupling type NGIICLZ is used in pairs or in conjunction with coupling type GIICLZ. The external teathed hubs are connected with intermediate shaft and the two coupling halves at both ends are connected with machine shaft and prime mover shaft respectively. Since the brake wheel is connected with coupling half the weight of brake wheel and braking load as well as vibration are borne fully by the coupling half. This improves meshing performance of crowned teeth during braking. Therefore, it is specially suitable for application with heavy load. Transmitted nominal torque ranges from 400 to 112000Nm.

To enhance lubrication sealing effect, to reduce number of parts and increase running reliability it is specially recommended to use integrated structure of sealed end cover with internal gear ring.

Parameters of brake wheel are shown in Table 16.

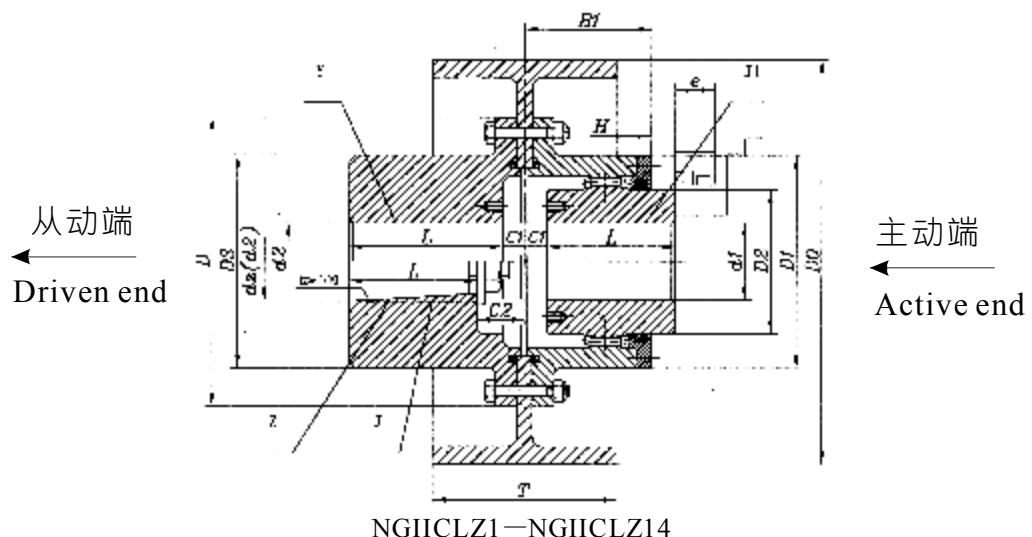


图14 NGIICLZ型鼓形齿式联轴器结构型式
Fig.14 Construction of the curved tooth coupling NGIICLZ

Y-Y型轴孔 J-J型轴孔 J₁-J₁型轴孔 Z-Z型轴孔
Y-Bore Type Y J-Bore Type J J₁-Bore Type J₁ Z-Bore Type Z

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表15 中间套的质量和转动惯量

型号 Type	中间套最小长度 Min Length S _{min} (mm)	质量 Mass m (kg)	转动惯量 Inertia I (kg · m ²)	中间套每加长10mm的质量 Mass of every add. 10 mm length m (kg)	中间套每加长10mm的转动惯量 inertia of every add. 10 mm length I (kg · m ²)
GICLT 01	75	1.58	0.0032	0.088	0.00011
GICLT 02	80	2.68	0.082	0.13	0.00022
GICLT 03	80	3.5	0.015	0.16	0.00041
GICLT 04	100	5.2	0.032	0.20	0.0008
GICLT 05	100	6.0	0.048	0.23	0.0012
GICLT 06	100	6.6	0.061	0.26	0.0017
GICLT 07	120	10.5	0.134	0.32	0.0030
GICLT 08	120	11.5	0.164	0.32	0.0039
GICLT 09	155	15.7	0.25	0.42	0.0045
GICLT 10	155	22.2	0.46	0.46	0.0064
GICLT 11	175	28.5	0.82	0.52	0.0091
GICLT 12	205	37.3	1.21	0.71	0.015
GICLT 13	205	46.4	1.61	0.83	0.024
GICLT 14	240	6.4	3.49	0.96	0.037
GICLT 15	240	73.4	4.42	1.03	0.044
GICLT 16	280	98.1	7.02	1.50	0.072
GICLT 17	280	132.4	10.78	2.50	0.16
GICLT 18	350	177.6	17.2	2.76	0.22
GICLT 19	350	182.4	19.3	2.96	0.27
GICLT 20	350	203.1	24.7	3.16	0.32
GICLT 21	350	261.6	38.1	4.20	0.50
GICLT 22	400	304.6	47.1	4.47	0.59
GICLT 23	400	391.6	70.9	5.77	0.88
GICLT 24	400	412	78.0	6.07	1.02
GICLT 25	400	450	84.2	6.33	1.16
GICLT 26	400	495	92.9	6.61	1.31
GICLT 27	500	560	106.6	8.01	1.66
GICLT 28	500	600	113.2	8.42	1.80
GICLT 29	500	650	120.7	8.75	1.94
GICLT 30	500	700	129.0	9.14	2.07

表 16 制动轮的主要尺寸、质量和转动惯量

Table 16 dimensions,Mass and Inertia of Brake Wheel

制动轮直径 Diameter D ₀ (mm)	制动轮宽度 Wide T (mm)	幅板厚度 Thickness K (mm)	质量 Mass m (mm)	转动惯量 Inertia I (kg · m ²)	制动轮直径 Diameter D ₀ (mm)	制动轮宽度 Wide T (mm)	幅板厚度 Thickness K (mm)	质量 Mass m (mm)	转动惯量 Inertia I (kg · m ²)
160	70	6	2.83	0.014	500	210	18	56.3	3.07
200	85	8	5.2	0.043	630	265	22	101.3	8.55
250	105	10	10.1	0.128	710	300	22	145.8	15.52
315	135	12	17.2	0.354	800	340	26	203	26.76
400	170	14	33.4	1.11					

表 17 制动盘的主要尺寸、质量和转动惯量

Table 17 dimensions,Mass and Inertia of Brake Disk

制动盘直径 Diameter D ₀ (mm)	制动盘厚度 Thickness T (mm)	质量 Mass m (mm)	转动惯量 Inertia I (kg · m ²)	制动盘直径 Diameter D ₀ (mm)	制动盘厚度 Thickness T (mm)	质量 Mass m (mm)	转动惯量 Inertia I (kg · m ²)
315	30	17.0	0.232	630	30	77.6	4.12
355	30	22.8	0.384	710	30	93.6	6.64
400	30	30.4	0.64	800	30	135.6	11.74
450	30	39.4	1.10	900	30	173.2	18.6
500	30	50.0	1.66	1000	30	193.2	26.1
560	30	61.4	2.56				

附录 Appendices

附表 1 动力机系数 K_w

动力机类型代号 Code of Prime mover	动力机名称 Description	动力机系数 Factor K_w	动力机类型代号 Code of Prime mover	动力机名称 Description	动力机系数 Factor K_w
I	电动机、透平 Motor & Turbine	1.0	III	二缸内燃机 I.C.E. with two Cylinders	1.4
II	四缸及四缸以上内燃机 Internal Combustion Engine with four and above Cylinders	1.2	IV	单缸内燃机 Single Cylinder Internal Combustion Engine	1.6

附表 2 起动系数 K_z

A.2 Starting Factor K_z

主动端起动频率 Z	≤ 120	$> 120 \sim 240$	> 240
起动系数 K_z	1.0	1.3	由制作厂确定 determined by manufacturer

附表 3 GICL型联轴器许用径向补偿量 Δ_y

A.3 Permissible Radial Compensation Value of Coupling Type GICL

(mm)

型号 Type	GICL 01	GICL 02	GICL 03	GICL 04	GICL 05	GICL 06	GICL 07	GICL 08
许用径向补偿量 Permissible radial compensation Δ_y	1.0	1.1	1.3	1.6	1.7	1.9	2.1	2.4
型号 Type	GICL 08	GICL 09	GICL 10	GICL 11	GICL 12	GICL 13	GICL 14	GICL 15
许用径向补偿量 Permissible radial compensation Δ_y	2.6	3.1	3.4	4.0	4.8	5.1	5.2	6.0
型号 Type	GICL 17	GICL 18	GICL 19	GICL 20	GICL 21	GICL 22	GICL 23	GICL 24
许用径向补偿量 Permissible radial compensation Δ_y	6.3	6.6	7.04	7.48	7.74	7.85	8.2	8.5
型号 Type	GICL 25	GICL 26	GICL 27	GICL 28	GICL 29	GICL 30		
许用径向补偿量 Permissible radial compensation Δ_y	8.8	9.0	9.0	9.6	9.6	9.9		

附表 4 GIICL型联轴器许用径向补偿量 Δ_y

A.4 Permissible Radial Compensation Value of Coupling Type GIICL

(mm)

型号 Type	GIICL 01	GIICL 02	GIICL 03	GIICL 04	GIICL 05	GIICL 06	GIICL 07	GIICL 08	GIICL 09
许用径向补偿量 Permissible radial compensation Δ_y	1.0	1.0	1.1	1.2	1.4	1.4	1.5	1.7	1.8
型号 Type	GIICL 10	GIICL 11	GIICL 12	GIICL 13	GIICL 14	GIICL 15	GIICL 16	GIICL 17	GIICL 18
许用径向补偿量 Permissible radial compensation Δ_y	2.0	2.1	2.3	2.6	4.5	4.8	5.3	5.4	5.8
型号 Type	GIICL 19	GIICL 20	GIICL 21	GIICL 22	GIICL 23	GIICL 24	GIICL 25		
许用径向补偿量 Permissible radial compensation Δ_y	5.8	6.4	6.6	6.8	8.0	8.4	8.5		

附表 5 联轴器工况系数 K

工作机名称 Description of Working Machinery			载荷类代号 Code of Load Type	工况系数 Duty Factor K
转向机构	Steering machine			
加煤机	Coal feder			
风筛	Pneumatic screen			1.20
装罐机械	Filler,Packing machine			
鼓风机 Blower	离心式	Centrifugal		
	轴流式	Axial		1.80
风扇 Fan	离心式	Centrifugal		1.20
	轴流式	Axial		1.80
泵 Pump	离心泵	Centrifugal		1.20
	回转泵 (齿轮泵、螺杆泵、滑片泵、叶形泵)	Rotating(gear,screw,vane pump)		1.80
压缩机 Compressor	离心式	Centrifugal		1.50
	轴流式	Axial		1.80
搅拌设备 Mixer	纯液体	Pure liquid		1.20
	液体加固体	liquid plus solid		
	液体可变密度	liquid with varied density		1.50
泵酿造和蒸馏设备 Brewing and distilling equipment	装瓶机械	Bottling machine		1.20
	过滤桶	Filter tank		1.50
均匀加载运输机 Evenly loaded conveyor	组装运输机	assembly for conveyor		
	带式输送机	belt conveyor		1.20
	斗式运输机	bucket conveyor		
	板式运输机	apron conveyor		
	链条式输送机	chain conveyor		
	链板式输送机	plate conveyor		
	箱式运输机	box type conveyor		1.50
不均匀加载机 Unevenly loaded conveyor	螺旋式运输机	crew conveyor		
	组装运输机	conveyor for assembly	I类 Type 1	
	带式输送机	belt conveyor		
	斗式运输机	bucket conveyor		1.80
	链条式输送机	chain conveyor		
给料机 Feeder	链板式输送机	plate conveyor		
	箱式运输机	box type conveyor		
	板式给料机	plate conveyor		1.50
	带式给料机	belt conveyor		
提升机械 Elevator	圆盘给料机	disc type feeder		
	螺旋给料机	dscrew feeder		
造纸设备 Paper machine	自动升降机	Automatic elevator		1.50
	重力卸料提升机	unloading by gravity		1.80
食品机械 Food processing equipment	漂白机	bleaching machine		1.20
	校平机	smoother		1.50
	卷取机	coiling machine		1.80
	清洗机	cleaning machine		
其他机床 other machine tools	瓶装罐装机械	botting machine and can filler		1.20
	谷类脱粒机	Grain sheller		1.50
其他机床 other machine tools	辅助传动装置	auxiliary transmission devices		1.50
	主传动装置	main drive unit		1.80

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续附表 5

工作机名称 Description of Working Machinery			载荷类代号 Code of Load Type	工况系数 Duty Factor K
纺织机械 Textile processing machine	开清棉机	cotton opener & beating machine		1.20
	定量给料机	feeder with constant amount		
	印花机	printing machine		
	浆纱机	sizing machine		
	染色机	dyeing machine		1.50
	压光机	roling press		
	起毛机	flubbing machine		
	压榨机	mangle		
	轧光机	planishing mill		
	棉花精整机（清洗、拉幅、碾压机等）	finishing equipment (cleaning, tenter,grounding)		
	黄化机	yanthator		
	罐蒸机	Pot evaporator		
	织布机	loom		
	梳理机	carding machine		
	卷取机	coiling machine		
印刷机械 Printing machine			I类 Type I	
废水处理设备 Waste water processing equipment	网筛	mesh screen		
	化学处理设备	chemical processing equipment		
	环形集尘器	ring type dust arrester		
	脱水筛	dewatering screen		
	砂粒集尘器	send dust arrester		
	废渣破碎机	Waste residue crusher		1.50
	快慢搅拌机	rapid and slow mixer		
	污泥收集器	muck collector		
	浓缩机	decker machine		
	真空过滤器	vacuum filter		
流动水进料网滤器 Flowing water feeding strainer mesh				
石油机械冷却设备 cooling equipment for petroleum processing machine				
往复多缸式压缩机 Reciprocating compressor with multi-				
通风机 Ventilator	冷却塔式	cooling tower type		2.40
	引风机（无风门控制）	air attraction type(without throttle control)		
泵 Pump	三缸或多缸单动活塞泵	three or multi-cylinder single action piston pump		2.10
	搅拌机 Mixer	筒形搅拌机 barrel type mixer		2.10
不均匀加载运输机 Unevenly load conveyor	混凝土搅拌机	concrete mixer		1.80
	板式运输机	apron conveyor		2.10
	螺旋式运输机	screw type conveyor		
提升机械 Elevator	往复式运输机	Reciprocating conveyor		1.80
	离心式卸料机	centrifugal unloader		3.00
	料斗式提升机	bucket elevator		1.80
石油机械 Petroleum machine	普通货车用提升机	bucket elevator		2.40
	石蜡过滤机	paraffin filtering machine		2.10
	油井泵	oil well pump		
	旋转窑	rotary kiln		2.40

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续附表 5

	工作机名称 Description of Working Machinery	载荷类代号 Code of Load Type	工况系数 Duty Factor K
造纸设备 Paper machine	卷绕机 coiler		1.80
	搅拌器和破碎机 mixer and crusher		
	叠层机 paper device		
	卷筒装置 drying machine		2.10
	烘干机 rolling mill		
	吸入滚轧机 hydraulic peeling machine		
	液压式剥皮机 mechanical peeling machine		
	机械式剥皮机 roling press		
	压光机 cutting machine bander		
	切断机 bander		2.40
食品机械 Food processing machine	打捆机 tinker tractor		
	原木拖运机		
	压力机 press		
	压皮滚筒 (齿轮传动) press drum(gear transmission)		2.70
木材加工机械 tinker processing equipment	甜菜切割机 feet cutting machine		
	搅面机 flour and water mixer		2.10
	绞肉机 meat chopper		
	甘蔗切割机 cxan cuter		2.40
工具机 Machine tool	分料机 stock sorter	II类 type II	1.80
	板坯运输机 conveyor for plate blank		
	刨床进给装置 feeding device of planer		
	刨床传动装置 transmission ofplaner		2.10
	剪切机进给装置 feeding device of shearer		
	剥皮机 (筒形) peeler(drum type)		
	修边机 edge trimmer		
	传动辊装置 edge trimmer		
	拖木机 (倾斜式) timber tractor(inclined)		
	拖木机 (竖式) timber tractor(vertical)		
橡胶机械 Rubber machine	送料辊装置 feeding roller gear		2.40
	刨床 planer		1.80
	弯曲机 bending machine		
	冲压机 (齿轮驱动装置) punching machine(grearing)		2.40
	攻丝机 tapping machine		3.00
轧制设备 Roling equipment	橡胶压延机 rubber rolling machine		
	压片机 enclosed freeger		2.40
	胶料粉碎机 rubber tyred forming mill		
	密闭式冷冻机		2.70
	轮胎式成型机		3.00
	纵剪切机 longitudinal Shearer		1.80
	绕线机 enwinding machine		2.10
	拉拔机小车架 Main drive of drwing machine		
	拉拔机主传动		
	成型机		2.40
	拉线机和压延机		
	不可逆输送辊道 forming machine		2.70

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续附表 5

工作机名称 Description of Working Machinery			载荷类代号 Code of Load Type	工况系数 Duty Factor K
水泥设备 Cement equipment	水泥窑	cement kiln		2.40
	干燥机和冷却机	drying and cooling machine		
	烘干机	drying stove		
	砂石粉碎机	Gravel crusher		
	棒式粉碎机	bar type crusher		
	旋转式粉碎机	rotary crusher		
	滚筒式粉碎机	drum type crusher		
起重机和卷扬机 Crane and hoist	球磨机	ball mill		2.70
	斜坡式卷扬机	inclined hoist		1.80
	抓斗起重机	grab crane		
	吊钩起重机	hook crane		
	桥式起重机	bridge crane		
	主卷扬机	main hoist winch		
	可逆式卷扬机	reversible hoist		2.40
绞车 (卷扬机)	Hoist winch		II类 type II	2.10
粘土加工设备	Clay processing equipment			2.40
球团机 (压坯机械)	blank rolling machine(pelleter)			
拖拉式卸货机 (间断负载)	Towed type unloader(intermittent load)			
挖泥机 Dredger	运输机	Conveyor		1.80
	通用绞车	universal winch		
	电缆盘装置	cable drum		
	机动绞车	powered winch		
	泵	pump		
	网筛传动装置	mesh screen drive		
	堆积机	stacker		
洗衣机 Washing machine	切割头传动装置	cutter head transmission		2.70
	夹具传动装置	fixture transmission		
	可逆式洗衣机	Reversible Mashing machine		
	滚筒式洗衣机	barrel type washing machine		
锤式粉碎机	Hammer-type crusher			2.40
旋转式筛石机	Rotary gravel screen-machine			
摆动运输机	Swinging conveyor			
破碎机 Breaker	碎矿机		III类 type III	3.30
	碎石机			
往复式给料机	Reciprocating feeder	ore breaker		3.00
可逆输送辊道	Reversible table roller	rock breaker		
重型机械 Heavy duty machine	初轧机	blooming mill		
	中厚板轧机	rolling mill for plate of medium and large thickness		
	机架辊	machine frame roller		
	剪切机	Shearer		
	冲压机	punching machine		

新一代联轴器

为您创造更多效益!

NOVEL COUPLINGS BRING MORE BENEFITS FOR YOU!



JS蛇形弹簧联轴器
JS steelflex tapered grid coupling



GICL、GIICL齿形齿式联轴器
GICL&GIICL curved tooth coupling



DG齿面鼓形齿式联轴器
DG curved tooth drum coupling



ML、MLPK梅花形弹性联轴器
ML&MLPK jaw coupling



JM、JM1J膜片联轴器
JM&JM1J disc(diaphragm) coupling



PGCLK鼓形齿式联轴器
PGCLK curved tooth drum coupling



SWP、SWO十字轴式万向联轴器
SWP&SWO carden universal coupling



AQ系列钢球式安全联轴器
AQ steel-ball safety coupling



新一代联轴器用于ZPMC气动轨道式集装箱龙门起重机在天津港 (2003年)
Novel couplings are used on ZPMC RMGS at the port of TIANJIN (2003)

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